

they move together as a single structure.

Powered side belts **106** and **107** are adjustably provided depending downwardly from the substantially horizontal legs of the L-frame members **156** and **157**. Conventional resettable clamp mechanisms are provided to make such an adjustable connection. Thus, the powered side belts **106** and **107** can be adjusted according to the position of the load path **101** and the width of the load **L** which is to pass along the load path **101**.

An intermediate supporting structure **158** is also provided which is connected with the L-frame members **156** and **157**. The intermediate supporting structure **158** further movably supports a shaft support and bearing structure **158A** which in turn supports the rotating shaft **112** over the load path **101**. The support and bearing structure **158A** is vertically adjustably connected to the intermediate supporting structure **158** by a lead screw arrangement **160** which can be driven by a crank **159**. The lead screw **160** is fixed in axial position, so that rotation thereof reacts with a non-rotational threaded portion of the support and bearing structure **158A** to thereby cause the support and bearing structure **158A** to move vertically along the lead screw **160**. Thus, the rotating shaft **112** is vertically adjustable with regard to the L-frame members **156** and **157**, which are themselves vertically adjustable with regard to the remainder of the supporting apparatus **154**, as discussed above. By this, not only are the powered side belts **106** and **107** adjustable horizontally and vertically, the rotating shaft **112** is vertically adjustable with respect to the powered side belts **106** and **107**.

The elements of the rotating shaft **112** are the same as that described above with regard to the FIG. 1 embodiment. Specifically, a lever **116** is provided including a pair of opposed arms to extend within the load path **101** for a particular load **L**. The appropriate sensors, pneumatic connections and control systems are also provided in a similar manner. Since the description and operation of these elements are exactly the same as that described above, no further explanation will be provided at this point for those features.

It is also preferable that the intermediate supporting structure **158**, the support and bearing structure **158A**, and the rotating shaft **112** be rotationally adjustable as a unit about an axis that extends perpendicular to the supports **109** and **110** and over the load path **101**. As shown in FIG. 20, this can be accomplished by providing a pivot pin **190** extending from the vertical portion of the intermediate structure **158** that is pivotally supported by a bearing surface provided on the vertical portion of the joint structure comprising L-frame members **156** and **157**. The pivot pin **190** can be conventionally secured to the bearing surface, such as by C-clips and the like. Moreover, the rotational connection may include means for locking the intermediate structure **158** relative to the L-frames **156** and **157** in multiple positions. Such locking means can comprise any of known friction lock, detent lock or the like mechanisms.

Another manner of providing such a rotational adjustment is illustrated in FIGS. 21 and 22. This manner is further advantageous in that the L-frame members **156** and **157** are not only vertically adjustable with respect to the supporting frame **154**, they are also rotatable as a unit about the axis extending perpendicular to the supports **109** and **110** and extending over the load path **101**. Moreover, the side belts **106** and **107** are also adjustable with the L-frame members **156** and **157**. To do this, a plate **169** is operatively positioned between the uprights **109** and **110** and the L-frame members **156** and **157**. The plate **169** includes elements **175** and **176**

which are slidably engaged within the slots **S** provided on the uprights **109** and **110**. It is the plate **169** which is thus vertically adjustable with regard to the uprights **109** and **110** by a conventional adjustment mechanism. The L-frame members **156** and **157** is further connected by a back plate **180** which connects between the substantially vertical portions of the L-frame members **156** and **157**.

Between the backing plate **180** and the plate **169**, a pivot pin **182** and bearing structure **170** are provided so that the backing plate **180**, and thus the L-frame members **156** and **157**, are pivotal about the plate **169**. Guide pins **171** and **172** are also provided extending from the backing plate **180** to engage within slots **173** and **174** of the plate **169**. The slots **173** and **174** define the pivotal limits of the L-frame members **156** and **157** about the pivot pin **182**. The backing plate **180** and the plate **169** can be conventionally locked in pivotal positions with respect to one another by any conventional locking means that may be provided integral with the pivot pin **182** or on either or both of the pins **171** and **172**. This arrangement allows the apparatus **108** of the present invention to be inclined to follow a load path **101** that is not parallel to the floor on which the apparatus is located.

Again, the rotating shaft **112** and its drive motor **113**, a belt **114**, and a brake-clutch **115**, as described above are all supported from the support and bearing assembly **158A** so as to be movable together with one another. Moreover, the sensor mechanisms such as shown in FIGS. 23-25 are also provided on the rotating shaft **112**.

With regard to the sensor controlling the cylinders **132** for the pins **130** and **131**, a similar sensor as that shown in FIG. 24 and described above is utilized. With regard to the sensor for controlling the cylinders within the lever **116** for cutting and extending the bodies thereof, a similar sensor as that illustrated in FIG. 25 and described above is also utilized. However, for controlling the brake-clutch mechanism **115**, it is contemplated to use a sensing mechanism such as that illustrated in FIG. 23 for controlling the activation and reactivation of the clutch **115** and the braking thereof.

Specifically, the cam **168** includes two lobes **168A** and **168B** which are sensed by a single inductive sensor **167**. This sensing means is a substitute for the sensor **33** described above which triggers the clutch on and off for moving the lever **16** through a rotation of approximately 180°. In this case, both of the lobes **168A** and **168B** when sensed by the inductive sensor **167** deactivate the clutch of the mechanism **115** and activate the braking of shaft **112**. When a load **L** hits the lever **116**, the one of the lobes **168A** and **168B** which is directly in front of the inductive sensor **167** is rotated past the sensor **167** and the clutch of the mechanism **115** is activated and the shaft **112** is rotated by the motor **113**. Such rotation continues for approximately 180° (or 360° for the one arm version) until the next of the lobes **168A** or **168B** moves to a position directly in front of the inductive sensor **167**.

It is understood that many other variations and embodiments for the present invention are possible which are within the scope of the present invention. Any of the adjustable mechanisms of the subject apparatus could be power driven or manually driven. Moreover, many other types of sensor mechanisms can be utilized for triggering and controlling the operation described above.

I claim:

1. Apparatus for applying adhesive handles comprising a leading portion and a trailing portion to loads driven along a path, said apparatus comprising:

a lever having at least one arm in a position across the